

ENCE 353 Midterm 2, Open Notes and Open Book

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**Exam Format and Grading.** Attempt all three questions. Partial credit will be given for partially correct answers, so please **show all of your working.**

Question	Points	Score
1	15	
2	15	
3	10	
Total	40	

Question 1: 15 points

**Analysis of a Supported Cantilever Beam Structure.** Figure 1 is a front elevation view of a cantilever beam carrying two external loads  $P$ .  $EI$  is constant along the cantilever beam.

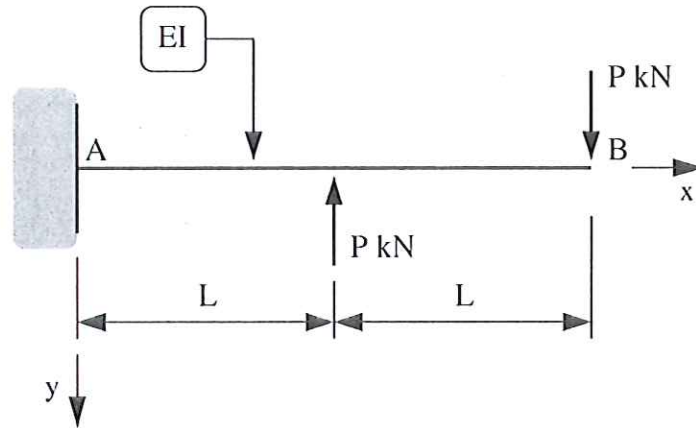


Figure 1: Cantilever beam carrying two applied loads  $P$  (kN).

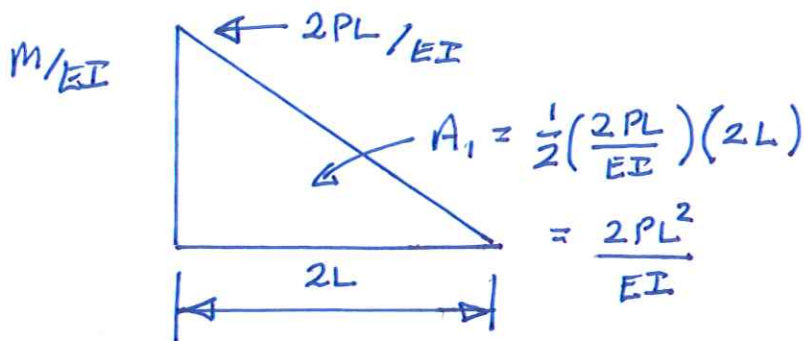
[1a] (3 pts) Briefly explain how the principle of superposition can be applied to this problem.



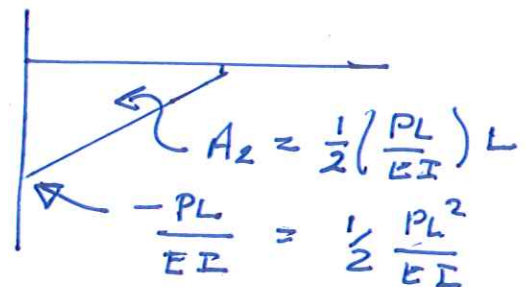
[1b] (5 pts) Use the method of moment-area to show that the clockwise rotation of point B is:

$$\theta_B = \left[ \frac{3}{2} \right] \frac{PL^2}{EI} \quad (1)$$

For system (a)



For system (b)



2

Rotation  $\theta_B = A_1 + (-A_2) = \frac{3}{2} \frac{PL^2}{EI} \quad \checkmark$

[1c] (7 pts) Use the method of moment-area to show that the vertical displacement at B is:

$$y(2L) = \left[ \frac{11}{6} \right] \frac{PL^3}{EI} \quad (2)$$

$y(2L) =$  first moment of area between B - A,  
evaluated about B.

For system (a)

$$y_a(2L) = A_1 \cdot \bar{x}_1 \quad \bar{x}_1 = \frac{2}{3}(2L) = \frac{4L}{3}$$

$$A_1 = \frac{2PL^2}{EI}$$

$$= \frac{2PL^2}{EI} \left( \frac{4}{3}L \right) = \frac{8}{3} \frac{PL^3}{EI} \downarrow$$

For system (b)

$$y_b(2L) = A_2 \bar{x}_2 \quad \bar{x}_2 = L + \frac{2}{3}L = \frac{5}{3}L$$

$$A_2 = \frac{1}{2} \frac{PL^2}{EI}$$

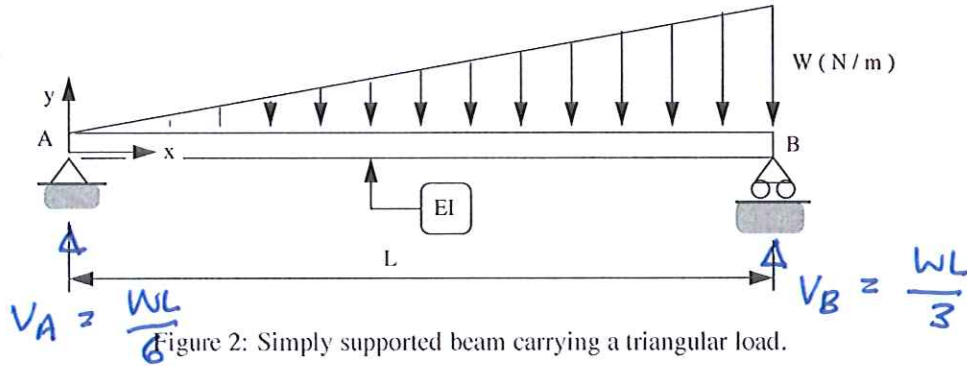
$$= \frac{1}{2} \frac{PL^2}{EI} \left( \frac{5}{3}L \right)$$

$$= \frac{5}{6} \left( \frac{PL^3}{EI} \right) \uparrow$$

$$\text{Net displacement } y(2L) = \left( \frac{8}{3} - \frac{5}{6} \right) \frac{PL^3}{EI} = \frac{11}{6} \frac{PL^3}{EI} \quad \checkmark$$

Question 2: 15 points

**Elastic Curve for Beam Deflections.** Figure 2 is a front elevation view of a simply supported beam that carries a triangular load.



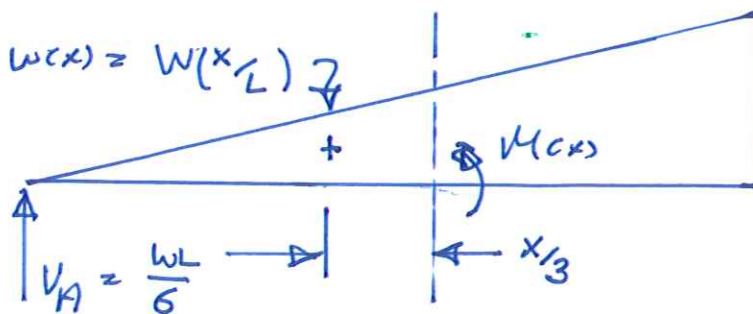
The load increases from zero at point A to  $W$  (N/m) at point B. Thus, the total beam loading is  $WL/2$ .

[2a] (5 pts). Starting from first principles of engineering (i.e., equilibrium of a substructure extracted from Figure 2), show that the bending moment at point  $x$  is:

$$M(x) = \left[ \frac{W}{6L} \right] x (L^2 - x^2). \quad (3)$$

From statics  $V_A = \frac{WL}{6}$  &  $V_B = \frac{WL}{3}$

Now consider moment at an arbitrary position  $x$ .



$$M(x) = \left( \frac{WL}{6} \right) x - \frac{1}{2} \left( \frac{Wx^2}{L} \right) \cdot \left( \frac{x}{3} \right) = \left( \frac{W}{6L} \right) x (L^2 - x^2).$$

[2b] (5 pts). Show that the elastic curve for beam deflection is given by (notice that in Figure 2, the y axis is pointing upwards):

$$y(x) = \left[ \frac{-W}{6LEI} \right] \left[ \frac{L^2 x^3}{6} - \frac{x^5}{20} - \frac{14L^4 x}{120} \right] \quad (4)$$

$$\frac{d^2 y}{dx^2} = \frac{-M(x)}{EI} \Rightarrow EI \frac{d^2 y}{dx^2} = \frac{-W}{6L} (L^2 x - x^3)$$

Integrating twice & applying boundary conditions.

$$y(0) = 0 \rightarrow B = 0$$

$$y(L) = 0 \rightarrow A = \left( \frac{L^5}{20} - \frac{L^5}{6} \right) \frac{1}{L} = \frac{-14}{120} L^4.$$

$$\Rightarrow y(x) = \left( \frac{-W}{6EIL} \right) \left[ \frac{L^2 x^3}{6} - \frac{x^5}{20} - \frac{14L^4}{120} \right]. \quad \checkmark$$

[2c] (5 pts). Show that the maximum beam curvature occurs at  $x = L/\sqrt{3}$ .

$$\phi = \frac{M(x)}{EI}, \quad \text{Max } \phi \Rightarrow \frac{dM}{dx} = 0$$

$$\Rightarrow L^2 - 3x^2 = 0 \Rightarrow x = \frac{L}{\sqrt{3}}.$$

Question 3: 10 points

**Simple Three-Pinned Arch.** Figure 3 is a front elevation view of a simple three-pinned arch that carries a total snow loading of  $3WL$  uniformly distributed over its upper section.

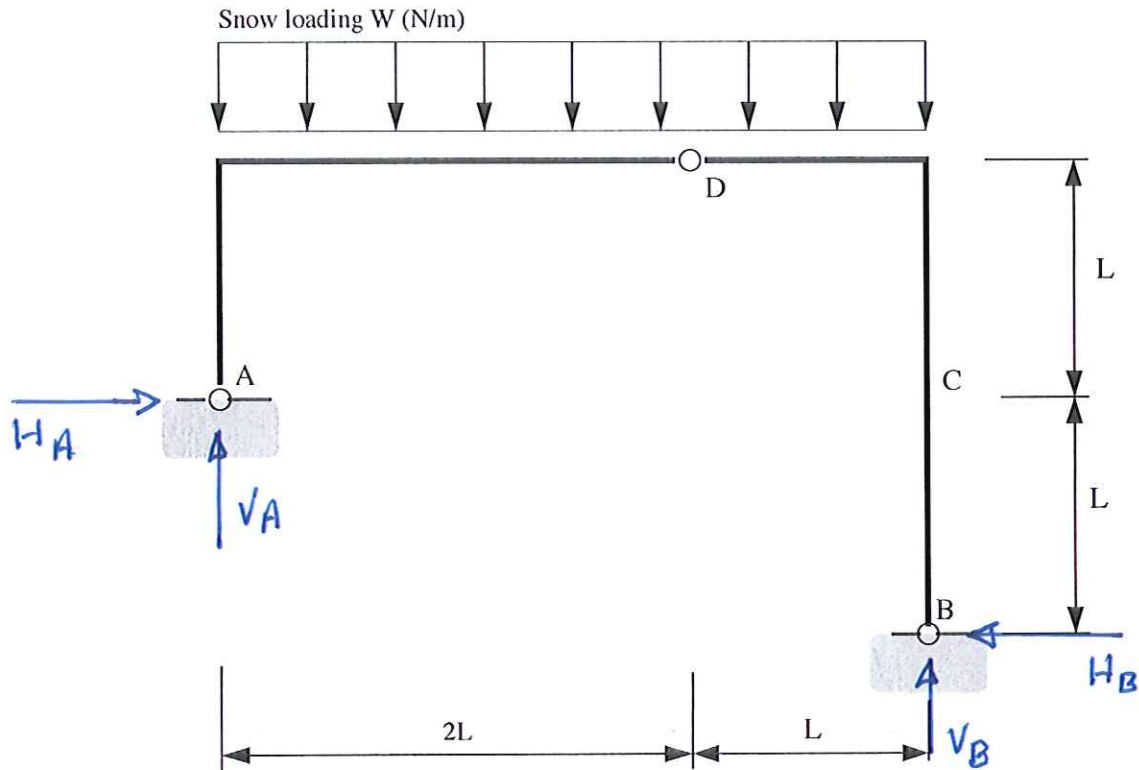


Figure 3: Front elevation view of a three-pinned arch that supports a snow loading.

[3a] (6 pts) Compute the vertical and horizontal components of reaction force at supports A and B as a function of  $W$  and  $L$ .

$$\sum V = 0 \rightarrow V_A + V_B = 3WL. \quad \text{--- (A)}$$

$$\sum H = 0 \rightarrow H_A = H_B \text{ (not useful)}. \quad \text{--- (B)}$$

$$\begin{aligned} \sum M_D = 0 \text{ (LHS)}. \quad (2WL)L + H_A \cdot L &= 2L V_A \\ \rightarrow 2WL + H_A &= 2V_A \quad \text{--- (C)} \end{aligned}$$

$$\begin{aligned} \sum M_D = 0 \text{ (RHS)} \quad (WL)\left(\frac{L}{2}\right) + H_B(2L) &= V_B L \\ \rightarrow WL + 4H_B &= 2V_B. \quad \text{--- (D)}. \end{aligned}$$



Question 3a continued:

Add (C) + (D), insert (B)

$$2WL + H_A + WL + 4H_B = 2(V_A + V_B) = 6WL$$

$$\Rightarrow 3WL + 5H_A = 6WL$$

$$\Rightarrow H_A = H_B = \frac{3}{5}WL. \quad \text{--- (E)}$$

Plug (E) into (C) & (D)

$$V_A = \frac{13}{10}WL, \quad V_B = \frac{17}{10}WL.$$

Check equilibrium.

$$V_A + V_B = \left(\frac{13}{10} + \frac{17}{10}\right)WL = 3WL \quad \checkmark$$

[3b] (4 pts) Draw and label the bending moment diagram.

